Solution of Two-Dimensional Stokes Flow Problems Using Improved Singular Boundary Method

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Abstract. In this paper, an improved singular boundary method (SBM), viewed as one kind of modified method of fundamental solution (MFS), is firstly applied for the numerical analysis of two-dimensional (2D) Stokes flow problems. The key issue of the SBM is the determination of the origin intensity factor used to remove the singularity of the fundamental solution and its derivatives. The new contribution of this study is that the origin intensity factors for the velocity, traction and pressure are derived, and based on that, the SBM formulations for 2D Stokes flow problems are presented. Several examples are provided to verify the correctness and robustness of the presented method. The numerical results clearly demonstrate the potentials of the present SBM for solving 2D Stokes flow problems.

AMS subject classifications: 76D07, 76M25

Key words: Singular boundary method, origin intensity factor, Stokes flow, fundamental solution.

1 Introduction

The incompressible viscous flow in slow motion, known as Stokes flow, is a classical problem in fluid dynamics. It can be regarded as a subset of Navier-Stokes flows, and has been widely applied in the industry. For the numerical analysis of Stokes flow problem, three formulations are well-known: vorticity-stream vector function, vorticity-velocity approach and primitive variable (velocity-pressure) approach.

Many numerical methods have been applied to the solution of the Stokes equations, such as finite difference method (FDM) [1], finite element method (FEM) [2] and boundary element method (BEM) [3]. More recently, with the development of the meshless

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method, some other simpler and fast methods have appeared in the literature, particularly the method of fundamental solution (MFS) which has been used to solving Stokes flow problems [4–6]. Over the last decade, the main advantage of the MFS, namely the simple computational implementation, has been recognized. However, a fictitious boundary for the distribution of source points is required in the MFS, which enormously restricts its application in the practical project.

Some efforts have been put in the elimination of this drawback, and numerous techniques are developed. A comprehensive review with respect to these methods can be found in the [7–9]. In this work, we focus on the singular boundary method (SBM), recently proposed by Chen and his collaborators [8, 10, 11]. For the SBM, the source and collocation points are both distributed on the physical boundary, thus there is no need of fictitious boundary required in the MFS. But, the singularity of the fundamental solution is caused when the source point coincides with the collocation point. To isolate the singularity of the fundamental solution, the origin intensity factor is introduced, and thus how to determine the factor is the key issue of the SBM. Prior to this study, the SBM has since been successfully applied to potential [8] and elasticity problems [11], in which we find that the method has a well performance.

The object of this paper is to extend the SBM for solving Stokes flow problems. Different from our previous SBM solution of 2D elasticity problems [11], a new regularization technique can accurately remove the singularities of the fundamental solution and its derivatives, and consequently, the origin intensity factors can be determined directly without requiring sample nodes as in [11]. Furthermore, based on the corresponding origin intensity factors, the SBM formulations for the velocity, traction and pressure are established. The rest of this paper is organized as follows. In Section 2, the governing equation of the Stokes problems is introduced, and the origin intensity factors for the velocity, traction and pressure are derived. Section 3 provides four numerical examples to assess the performance of the proposed SBM scheme. Finally, some conclusions are provided in Section 4.

2   The SBM formulation for Stokes problems

In this paper, we always assume that Ω is a bounded domain in R^2, Ω^e is its open complement; $\Gamma = \partial\Omega$ denotes their common boundary; $\mathbf{n}(\mathbf{x})$ and $\mathbf{t}(\mathbf{x})$ are the unit outward normal vector and tangential vector of Γ to domain Ω at point $\mathbf{x}$, respectively.

Assuming the incompressibility for the fluid medium, the governing equation for the steady-state Stokes flow problems can be expressed as follows:

$$\text{Momentum equation:} \qquad -p_{,i} + \mu u_{i,jj} = 0, \quad \mathbf{x} \in \Omega, \qquad (2.1a)$$

$$\text{Continuity equation:} \qquad u_{i,i} = 0, \quad \mathbf{x} \in \Omega, \qquad (2.1b)$$

$$\text{Boundary condition:} \qquad \begin{cases} u_i = \bar{u}_i, & \mathbf{x} \in \Gamma_u, \\ t_i = \bar{t}_i, & \mathbf{x} \in \Gamma_t, \end{cases} \qquad (2.1c)$$

where u_i is the velocity, p the pressure, μ the coefficient of viscosity of the fluid, $\bar{u}_i$ and $\bar{t}_i$ the given values of velocity u_i and traction t_i on the boundary Γ_u and Γ_t ($\Gamma = \Gamma_u \cup \Gamma_t$) respectively, and $i,j = 1,2$. The stress σ_{ij} ($i,j=1,2$) in the fluid can be written as

$$\sigma_{ij} = -p\delta_{ij} + \mu(u_{i,j} + u_{j,i}), \quad \mathbf{x} \in \Omega, \tag{2.2}$$

and traction t_i ($i=1,2$) on Γ can be expressed as

$$t_i = \sigma_{ij} n_j, \quad \mathbf{x} \in \Gamma, \tag{2.3}$$

where n_j is the component of the outward normal vector $\mathbf{n}$.

Employing indicial notation for coordinates of points $\mathbf{x}$ and $\mathbf{s}$, i.e., x_1, x_2 and s_1, s_2, respectively, the fundamental solutions of the Stokes equation are given as [6]

$$U_{ij}(\mathbf{x},\mathbf{s}) = \frac{1}{4\pi\mu}\left[-\delta_{ij}\ln r(\mathbf{x},\mathbf{s}) + r_{,i}(\mathbf{x},\mathbf{s})r_{,j}(\mathbf{x},\mathbf{s})\right], \quad i,j=1,2, \tag{2.4a}$$

$$P_i(\mathbf{x},\mathbf{s}) = \frac{1}{2\pi}\frac{r_{,i}(\mathbf{x},\mathbf{s})}{r(\mathbf{x},\mathbf{s})}, \qquad\qquad i=1,2, \tag{2.4b}$$

where $r(\mathbf{x},\mathbf{s})$ is the Euclidean distance between the collocation point $\mathbf{x}$ and source point $\mathbf{s}$,

$$r_{,i}(\mathbf{x},\mathbf{s}) = \frac{\partial r(\mathbf{x},\mathbf{s})}{\partial x_i} = \frac{x_i - s_i}{r(\mathbf{x},\mathbf{s})}$$

expresses the derivative of the distance r with respect to x_i, and δ_{ij} is the Kronecker symbol.

Making use of the Eqs. (2.2)-(2.4b), the fundamental solutions of the traction and stress can be respectively expressed as

$$T_{ij}(\mathbf{x},\mathbf{s}) = -\frac{1}{\pi r}r_{,i}(\mathbf{x},\mathbf{s})r_{,j}(\mathbf{x},\mathbf{s})r_{,k}(\mathbf{x},\mathbf{s})n_k(\mathbf{x}), \qquad i,j,k=1,2, \tag{2.5a}$$

$$D_{ijk}(\mathbf{x},\mathbf{s}) = -\frac{1}{\pi r}r_{,i}(\mathbf{x},\mathbf{s})r_{,j}(\mathbf{x},\mathbf{s})r_{,k}(\mathbf{x},\mathbf{s}), \qquad i,j,k=1,2. \tag{2.5b}$$

The SBM uses the fundamental solution as the basis function of its approximation, which is similar to the MFS [7, 12–14]. However, unlike the MFS, the collocation points and source points of the SBM are coincident and placed on the physical boundary, as seen in Figs. 1(a) and (b). For the SBM, we have a fundamental assumption [11, 15] which is the existence of the origin intensity factor upon the singularity of the coincident source-collocation nodes for mathematically well-posed problems.

With the help of the above assumption, the SBM interpolation formulations for 2D
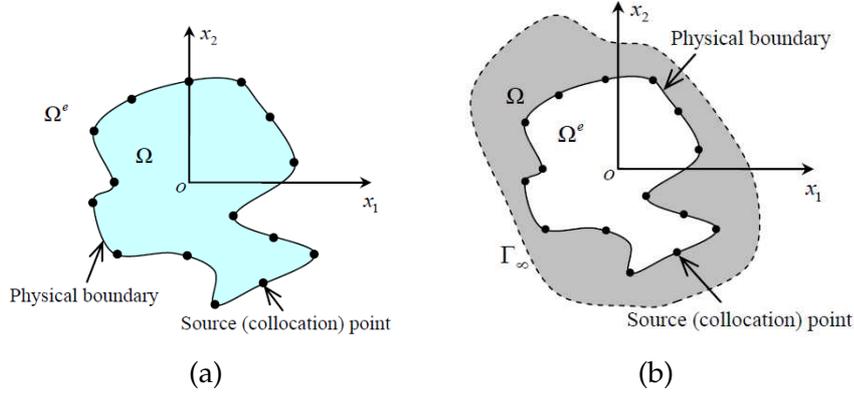
Figure 1: The source (collocation) point distributions: (a) for interior problems, (b) for exterior problems.

Stokes flow problems can be expressed as

$$u_i(\mathbf{x}^m) = \sum_{n=1, n \neq m}^{N} \alpha_j(\mathbf{s}^n) U_{ij}(\mathbf{x}^m, \mathbf{s}^n) + \alpha_j(\mathbf{s}^m) u_{ij}(\mathbf{x}^m), \qquad (2.6a)$$

$$t_i(\mathbf{x}^m) = \sum_{n=1, n \neq m}^{N} \alpha_j(\mathbf{s}^n) T_{ij}(\mathbf{x}^m, \mathbf{s}^n) + \alpha_j(\mathbf{s}^m) t_{ij}(\mathbf{x}^m), \qquad (2.6b)$$

$$p(\mathbf{x}^m) = \sum_{n=1, n \neq m}^{N} \alpha_j(\mathbf{s}^n) P_j(\mathbf{x}^m, \mathbf{s}^n) + \alpha_j(\mathbf{s}^m) p_j(\mathbf{x}^m), \qquad (2.6c)$$

where $\alpha_j(\mathbf{s}^n)$ is the unknown coefficient, $\mathbf{x}^m$ and $\mathbf{s}^n$ are the mth collocation point and the nth source point, respectively. $u_{ij}(\mathbf{x}^m)$, $t_{ij}(\mathbf{x}^m)$, $p_j(\mathbf{x}^m)$ are defined as the origin intensity factors, namely diagonal and sub-diagonal elements of the SBM interpolation matrix.

2.1   The calculation of $u_{ij}(\mathbf{x}^m)$

As the collocation point $\mathbf{x}^m$ approaches to the source point $\mathbf{s}^n$, the distance between these two boundary nodes trends to zero, which would cause different orders of singularities in Eqs. (2.6a)-(2.6c). By adopting an average value of the fundamental solution over a portion of the boundary, the calculated formulation of $u_{ij}(\mathbf{x}^m)$ can be expressed as

$$
\begin{aligned}
u_{ij}(\mathbf{x}^m) &= \frac{1}{L_m} \int_{\Gamma_m} U_{ij}(\mathbf{x}^m, \mathbf{s}) d\Gamma(\mathbf{s}) \\
&= \frac{1}{4\pi\mu L_m} \left\{ -\delta_{ij} \int_{\Gamma_m} \ln r(\mathbf{x}^m, \mathbf{s}) d\Gamma(\mathbf{s}) + \int_{\Gamma_m} r_{,i}(\mathbf{x}^m, \mathbf{s}) r_{,j}(\mathbf{x}^m, \mathbf{s}) d\Gamma(\mathbf{s}) \right\},
\end{aligned}
\qquad (2.7)
$$

where L_m indicates the half distance between the source point $\mathbf{s}^{m-1}$ to the source point $\mathbf{s}^{m+1}$.

Obviously, the second integral at the right side of Eq. (2.7) has no singularity, and thus it can be accurately calculated by using the standard Gaussian quadrature. But, the following special treatment is required for the first integral which has the weak singularity

$$
\int_{\Gamma_m} \ln r(\mathbf{x}^m, \mathbf{s}) d\Gamma(\mathbf{s}) = \int_{-1}^1 J(\xi) \ln r(\mathbf{x}^m, \xi) d\xi
$$
$$
= \int_{-1}^1 [J(\xi) - J(\eta)] \ln r(\mathbf{x}^m, \xi) d\xi + J(\eta) \int_{-1}^1 \ln r(\mathbf{x}^m, \xi) d\xi, \qquad (2.8)
$$

in which ξ is the intrinsic coordinate that transforms the integral so that it is mapped onto the interval $[-1,1]$, $\eta \in [-1,1]$ represents the position of $\mathbf{x}^m$, and $J(\xi)$ denotes the Jacobian of the transformation. Now, the two integrals at the right side of Eq. (2.8) can be well calculated by using the standard Gaussian quadrature and analytical technique, respectively.

2.2   The calculation of $t_{ij}(\mathbf{x}^m)$

By using a subtracting and adding-back technique [16], the SBM traction formulation for interior problems can be re-expressed as follows

$$
t_i(\mathbf{x}^m) = \sum_{n=1}^N \alpha_j(\mathbf{s}^n) T_{ij}^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n) = \sum_{n=1, n \neq m}^N \left(\alpha_j(\mathbf{s}^n) - \frac{L_n}{L_m} \alpha_j(\mathbf{x}^m) \right) T_{ij}^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)
$$
$$
+ \frac{\alpha_j(\mathbf{x}^m)}{L_m} \sum_{n=1}^N \left[L_n T_{ij}^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n) + L_n T_{ij}^{\mathrm{E}}(\mathbf{s}^n, \mathbf{x}^m) \right], \qquad (2.9)
$$

where

$$
\sum_{n=1}^N L_n T_{ij}^{\mathrm{E}}(\mathbf{s}^n, \mathbf{x}^m) = 0, \qquad (2.10)
$$

and L_n is the half distance between source nodes $\mathbf{s}^{n-1}$ and $\mathbf{s}^{n+1}$, the superscript I and E denote the interior domain problem and the exterior domain problem, respectively. The derivations of Eq. (2.10) are given as in Appendix.

Based on the dependency of the outward normal vector on two kernel functions of interior and exterior problems [17], one can obtain the following relationship

$$
T_{ij}^{\mathrm{I}}(\mathbf{s}^n, \mathbf{x}^m) = \kappa T_{ij}^{\mathrm{E}}(\mathbf{s}^n, \mathbf{x}^m), \qquad (2.11)
$$

where κ is equal to 1 as $m = n$ and is equal to -1 as $m \neq n$.

Substituting (2.11) into (2.9), we have

$$
t_i(\mathbf{x}^m) = \sum_{n=1, n \neq m}^N \alpha_j(\mathbf{s}^n) T_{ij}^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n) - \frac{\alpha_j(\mathbf{x}^m)}{L_m} \left(\sum_{n=1, n \neq m}^N L_n T_{ij}^{\mathrm{I}}(\mathbf{s}^n, \mathbf{x}^m) - \chi_m^{\mathrm{I}} \right), \qquad (2.12)
$$

in which

$$\chi_m^{\mathrm{I}} = L_m T_{ij}^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^m) + L_m T_{ij}^{\mathrm{I}}(\mathbf{s}^m, \mathbf{x}^m). \tag{2.13}$$

The terms $T_{ij}^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)$ and $T_{ij}^{\mathrm{I}}(\mathbf{s}^n, \mathbf{x}^m)$ in Eq. (2.12) are regular and can be calculated directly because $\mathbf{x}^m$ and $\mathbf{s}^n$ are not coincident, i.e., $n \neq m$. The remaining term χ_m^{I} can be approximated as the following integral

$$\chi_m^{\mathrm{I}} \approx \int_{\Gamma_m} T_{ij}^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}) + T_{ij}^{\mathrm{I}}(\mathbf{s}, \mathbf{x}^m) d\Gamma(\mathbf{s}). \tag{2.14}$$

We can easily find that the integral (2.14) has no singularity, so it can be accurately calculated by using the standard Gaussian quadrature.

Using the procedure described above, the final form of the regularized traction equation for interior problems can be written as

$$t_i(\mathbf{x}^m) = \sum_{n=1, n \neq m}^{N} \alpha_j(\mathbf{s}^n) T_{ij}^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n) - \frac{\alpha_j(\mathbf{x}^m)}{L_m} \left(\sum_{n=1, n \neq m}^{N} L_n T_{ij}^{\mathrm{I}}(\mathbf{s}^n, \mathbf{x}^m) - \chi_m^{\mathrm{I}} \right) \tag{2.15}$$

and then the origin intensity factor can be calculated by the following formulation

$$t_{ij}(\mathbf{x}^m) = -\frac{1}{L_m} \left(\sum_{n=1, n \neq m}^{N} L_n T_{ij}^{\mathrm{I}}(\mathbf{s}^n, \mathbf{x}^m) - \chi_m^{\mathrm{I}} \right). \tag{2.16}$$

Using a similar derivation, the regularized traction equation for exterior problems can be easily constructed as

$$t_i(\mathbf{x}^m) = \sum_{n=1, n \neq m}^{N} \alpha_j(\mathbf{s}^n) T_{ij}^{\mathrm{E}}(\mathbf{x}^m, \mathbf{s}^n) + \frac{\alpha_j(\mathbf{x}^m)}{L_m} \left(\sum_{n=1, n \neq m}^{N} L_n T_{ij}^{\mathrm{E}}(\mathbf{s}^n, \mathbf{x}^m) + \chi_m^{\mathrm{E}} \right), \tag{2.17}$$

where

$$\chi_m^{\mathrm{E}} \approx \int_{\Gamma_m} T_{ij}^{\mathrm{E}}(\mathbf{x}^m, \mathbf{s}) + T_{ij}^{\mathrm{E}}(\mathbf{s}, \mathbf{x}^m) d\Gamma(\mathbf{s}). \tag{2.18}$$

Also, the origin intensity factor for exterior problems can be expressed as

$$t_{ij}(\mathbf{x}^m) = \frac{1}{L_m} \left(\sum_{n=1, n \neq m}^{N} L_n T_{ij}^{\mathrm{E}}(\mathbf{s}^n, \mathbf{x}^m) + \chi_m^{\mathrm{E}} \right). \tag{2.19}$$

In order to carry out integrations in Eqs. (2.8), (2.14) and (2.18), the boundary has to be discretised into boundary mesh. However, the numerical integration in the evaluation of origin intensity factors only increase negligible computing costs in terms of the total costs, while keeps essential meshless merit since mesh is only used for diagonal numerical integration.

2.3   The calculation of $p_j(\mathbf{x}^m)$

By adopting a subtracting and adding-back technique, the pressure equation (2.6c) for interior problem can be expressed as

$$p(\mathbf{x}^m) = \sum_{n=1,n\neq m}^{N} \left(\alpha_j(\mathbf{s}^n) - \frac{L_n}{L_m}\alpha_j(\mathbf{x}^m)\right) P_j^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n) + \frac{1}{L_m}\alpha_j(\mathbf{x}^m)\sum_{n=1}^{N} L_n P_j^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n), \qquad (2.20)$$

where L_n denotes the half distance of source points $\mathbf{s}^{n-1}$ and $\mathbf{s}^{n+1}$. Making use of the notation

$$\psi(\mathbf{x}^m,\mathbf{s}^n) = \frac{1}{2\pi}\ln r(\mathbf{x}^m,\mathbf{s}^n) \qquad (2.21)$$

and the following relationship

$$P_j^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n) = \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{x}^m)} n_j(\mathbf{x}^m) + \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{x}^m)} t_j(\mathbf{x}^m), \qquad (2.22)$$

the Eq. (2.20) can be rewritten as

$$\begin{aligned}
p(\mathbf{x}^m) = {} & \sum_{n=1,n\neq m}^{N} \left(\alpha_j(\mathbf{s}^n) - \frac{L_n}{L_m}\alpha_j(\mathbf{x}^m)\right) P_j^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n) \\
& + \frac{\alpha_j(\mathbf{x}^m)}{L_m}\left\{ n_j(\mathbf{x}^m)\sum_{n=1}^{N} \left(L_n\frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{x}^m)} + L_n\frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)}\right) \right. \\
& \left. + t_j(\mathbf{x}^m)\sum_{n=1}^{N} \left(L_n\frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{x}^m)} + L_n\frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)}\right) \right\},
\end{aligned} \qquad (2.23)$$

in which

$$\sum_{n=1}^{N} L_n\frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)} = 0, \quad \sum_{n=1}^{N} L_n\frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)} = 0. \qquad (2.24)$$

The derivations of Eq. (2.24) are provided in Appendix.

Based on the dependency of the outward normal or tangent vectors on two kernel functions of interior and exterior problems, we can obtain following relationships [17]:

$$\frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)} = \kappa\frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)}, \quad \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)} = \kappa\frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)}, \qquad (2.25)$$

in which κ is equal to 1 as $n=m$ and is equal to -1 as $n\neq m$. Besides, for arbitrarily smooth boundary, we assume that the source point $\mathbf{s}^n$ moves gradually close to the collocation

point $\mathbf{x}^m$ along a line segment, e.g., the tangential direction of $\mathbf{x}^m$, then we have

$$
\begin{cases}
\lim\limits_{\mathbf{s}^n \to \mathbf{x}^m} \dfrac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{x}^m)} + \dfrac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)} = 0, \\[3mm]
\lim\limits_{\mathbf{s}^n \to \mathbf{x}^m} \dfrac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{x}^m)} + \dfrac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)} = 0.
\end{cases} \tag{2.26}
$$

With the help of Eqs. (2.25) and (2.26), the finally regularized pressure equation can be obtained as follows

$$
\begin{aligned}
p(\mathbf{x}^m) = {}& \sum_{n=1, n \neq m}^{N} \alpha_j(\mathbf{s}^n) P_j^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n) \\
& - \frac{\alpha_j(\mathbf{x}^m) L_n}{L_m} \sum_{n=1, n \neq m}^{N} \left(n_j(\mathbf{x}^m) \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)} + t_j(\mathbf{x}^m) \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)} \right).
\end{aligned} \tag{2.27}
$$

It can be observed from the above equation that the origin intensity factor $p_j(\mathbf{x}^m)$ for interior problems can be calculated by using the formulation:

$$
p_j(\mathbf{x}^m) = -\frac{L_n}{L_m} \sum_{n=1, n \neq m}^{N} \left(n_j(\mathbf{x}^m) \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)} + t_j(\mathbf{x}^m) \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)} \right). \tag{2.28}
$$

With the similar derivation, the regularized pressure equation for exterior problems can be expressed as

$$
\begin{aligned}
p(\mathbf{x}^m) = {}& \sum_{n=1, n \neq m}^{N} \alpha_j(\mathbf{s}^n) P_j^{\mathrm{E}}(\mathbf{x}^m, \mathbf{s}^n) - \frac{\alpha_j(\mathbf{x}^m)}{L_m} \Big\{ n_j(\mathbf{x}^m) \\
& + \sum_{n=1, n \neq m}^{N} \left(L_n \frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)} n_j(\mathbf{x}^m) + L_n \frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)} t_j(\mathbf{x}^m) \right) \Big\},
\end{aligned} \tag{2.29}
$$

and the origin intensity factor is given as

$$
p_j(\mathbf{x}^m) = \frac{-1}{L_m} \Big\{ n_j(\mathbf{x}^m) + \sum_{n=1, n \neq m}^{N} \left(L_n \frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)} n_j(\mathbf{x}^m) + L_n \frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m, \mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)} t_j(\mathbf{x}^m) \right) \Big\}. \tag{2.30}
$$

2.4 The formulations of velocity and stress in the domain

Using the procedure described above, we have derived the SBM formulations for both the velocity and traction boundary conditions (2.1c). For the boundary value problem (2.1a)-(2.1c), unknown coefficients $a_j(\mathbf{s}^n)$ ($j = 1, 2$, $n = 1, \cdots, N$) can be determined by collocating N observation points on the boundary conditions. Once all boundary unknowns

are solved, the velocity and stress at any inner point $\mathbf{y}$ can be calculated by using the following formulations

$$u_i(\mathbf{y}) = \sum_{n=1}^{N} \alpha_j(\mathbf{s}^n) U_{ij}(\mathbf{y}, \mathbf{s}^n), \tag{2.31a}$$

$$\sigma_{ij}(\mathbf{y}) = \sum_{n=1}^{N} \alpha_k(\mathbf{s}^n) D_{ijk}(\mathbf{y}, \mathbf{s}^n). \tag{2.31b}$$

As the point $\mathbf{y}$ approaches the boundary, the above formulations will encounter the well-known boundary layer effect, as observed as well in the boundary element method. This drawback can be remedied by using the technique in paper [18]. In addition, the SBM formulation for 3D stokes problems can be obtained through the similar derivations. Some work along this line is already underway and will be reported in a subsequent paper.

3 Numerical examples

We present four numerical experiments to demonstrate the accuracy and efficiency of the proposed SBM formulations for solving 2D Stokes flow problems. An experimental program based on the present method is written in Compaq Visual FORTRAN 6.6. In all examples, the coefficient of viscosity is $\mu = 0.5$, Ndenotes the number of boundary nodes. In all numerical examples, the boundary nodes are uniformly distributed on the boundary unless a special description is given.

3.1 Flow caused with the uniform rotation of a circular cylinder

In this case, we consider the flow in an infinite 2D medium caused with the uniform rotation of a circular cylinder, as shown in Fig. 2. The radius of the cylinder is $R = 1$ and the angular velocity is $\hat{\omega} = 1$. The analytical solution [19] of this problem in the polar coordinate can be given as follows

$$u_r(r,\theta) = 0, \quad u_\theta(r,\theta) = \hat{\omega}R^2/r, \quad \sigma_{r\theta}(r,\theta) = -2\mu\hat{\omega}R^2/r^2 \quad \text{and} \quad P(r,\theta) = 0. \tag{3.1}$$

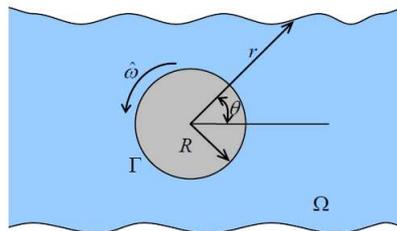The velocity is specified on the boundary using the above solution.

Figure 2: A cylinder with uniform rotation in an infinite domain.

Table 1: Results of the velocity u_θ and the shear stress $\sigma_{r\theta}$ at the point $(4,0)$.

N	Exact solution u_θ : 0.2500		Exact solution $\sigma_{r\theta}$: -0.0625	
	Numerical	Numerical (Non)	Numerical	Numerical (Non)
20	0.2518058E+00	0.2531024E+00	-0.6295145E-01	-0.6331438E-01
60	0.2506039E+00	0.2520482E+00	-0.6265097E-01	-0.6300217E-01
100	0.2503622E+00	0.2514168E+00	-0.6259055E-01	-0.6276409E-01
140	0.2502587E+00	0.2509977E+00	-0.6256467E-01	-0.6261948E-01
180	0.2502012E+00	0.2506284E+00	-0.6255029E-01	-0.6259028E-01
220	0.2501646E+00	0.2505215E+00	-0.6254114E-01	-0.6257649E-01
260	0.2501392E+00	0.2505029E+00	-0.6253481E-01	-0.6256601E-01
300	0.2501207E+00	0.2504913E+00	-0.6253017E-01	-0.6256158E-01

Table 1 lists the results of the velocity u_θ and the shear stress $\sigma_{r\theta}$ at the inner point $(4,0)$ by using the present method as the total number of boundary nodes varies from 20 to 300. As we can see, the numerical results remain in good agreement with the exact solutions. Furthermore, with an increasing number of boundary nodes, the present SBM performs stably with a fast convergence rate and the relative errors for u_θ and $\sigma_{r\theta}$ are both less than 5% as the number of the boundary nodes reaches up to 300. We also observe that the satisfied numerical results sill can be obtained by using non-uniformly distributed boundary nodes (the ratio of the boundary node numbers in four quadrants is $3:2:3:2$).

With using 40 boundary nodes, the field solutions of $u_\theta(r,0)$ and $\sigma_{r\theta}(r,0)$ at different points $(r,0)$ are plotted in the Fig. 3. Good match can be observed from the comparison between the numerical and exact solutions, which indicates that satisfied results can be obtained by the present SBM even though very few boundary nodes are distributed on the boundary. Table 2 shows the numerical results of the pressure p computed at boundary points in the angle interval $[0,0.5\pi)$ using the Eq. (2.29). Obviously, the SBM formulation (2.29) for the pressure passes the test successfully.
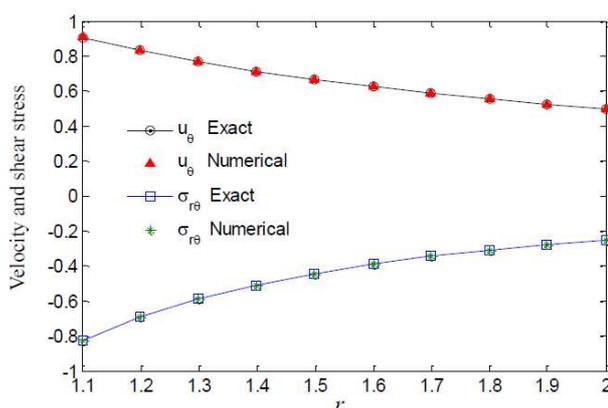

Figure 3: Velocity u_θ and shear stress $\sigma_{r\theta}$ at the different points $(r,0)$.

Table 2: Results of the pressure p at boundary points in the angle interval $[0,0.5\pi)$.

θ	Numerical	θ	Numerical
0.00π	-0.2037259E-13	0.25π	-0.1925196E-13
0.05π	-0.2223222E-13	0.30π	-0.2009504E-13
0.10π	-0.2082362E-13	0.35π	-0.2307182E-13
0.15π	-0.2034484E-13	0.40π	-0.1871420E-13
0.20π	-0.2247508E-13	0.45π	-0.2002218E-13
Exact	0.0000000E+00		

3.2 A recirculating flow in a circular cavity

We next tackle a model problem consisted of a recirculating flow in a circular cavity. The radius of the circular is assumed to be unity. The configuration and boundary conditions of the problem is depicted in Fig. 4. In the low half of the boundary, the velocity $u_\theta = u_r = 0$ are prescribed, and in the upper half, the velocity $u_\theta = 1$ and $u_r = 0$ are imposed.
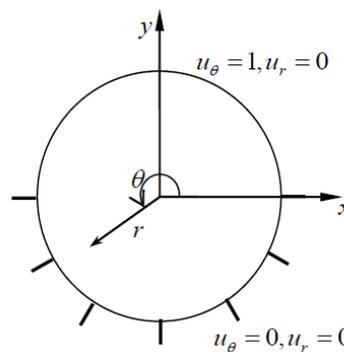
Figure 4: Circular cavity flow problem with boundary conditions.

In this model, there are 80 boundary nodes on the circular boundary. Fig. 5 shows the velocity vector plots within the fluid obtained by using analytical formulation [20] and the proposed SBM, where 800 points $\{(\rho,\theta)|\rho = i/21,\ \theta = (1+2j)\pi/40,\ i = 1,\cdots,20;\ j = 1,\cdots,40\}$ are distributed inside the domain. It can be seen that present computations give very satisfied results as compared with the exact solutions. We also can observed that the velocity becomes larger as the calculated point approaches to the upper half of boundary and reduces to zero as the calculated point approaches to the low half of boundary, which reflects the reasonable physical characteristics.

Fig. 6 shows the comparison of velocity profile on the vertical centerline, where the SBM solutions are compared with the exact solutions [20] and the numerical solutions obtained by the multiquadrics method (MQ) with 612 collocation points [21]. As can be seen in Fig. 6, the present SBM produces more accurate results than the MQ scheme in the computation of velocity u_θ, even if only 80 collocation points are used.
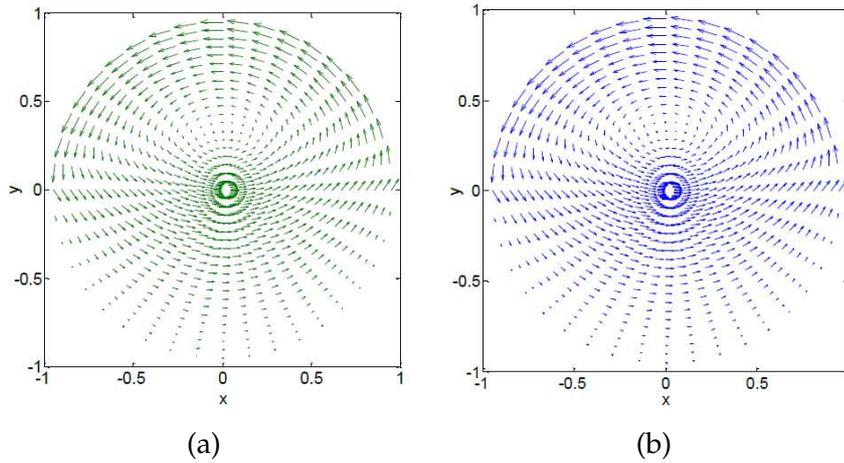
Figure 5: Vector plot of the velocity field for a circular cavity: (a) Exact solution, (b) SBM solution.
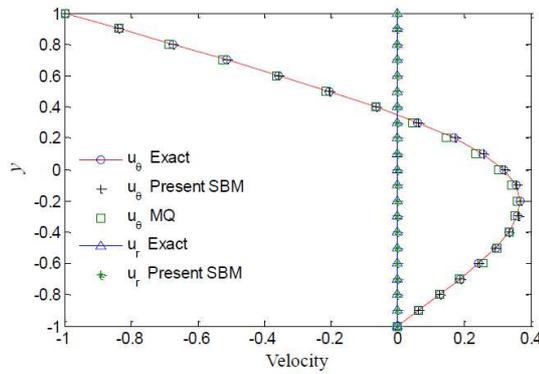

Figure 6: Velocity profile on the line $x=0$ for a circular cavity.

3.3   Shear flow between two parallel plates

In this case, the flow between two parallel plates is considered. The top plate is moving with a constant speed v_0 in the x-direction and no slip condition is assumed between the plates and fluid. The dimension of the model is shown in Fig. 7, and $H=1$, $L=3$. The analytical solution [19] for this problem is

$$u_x(x,y) = v_0 y/H, \quad u_y = 0, \tag{3.2}$$

where $v_0 = 10$.

The mixed boundary conditions are used in this case, so the SBM formulations (2.6a) and (2.6b) can both be tested. For the upper boundary, $u_x = v_0$ and $u_y = 0$ are specified, while for lower boundary, velocity are given as $u_x = u_y = 0$. For the two vertical boundaries, tractions are givens as $t_x = 0$, $t_y = \mu v_0/H$ at $x = L$; and $t_x = 0$, $t_y = -\mu v_0/H$ at $x = 0$.
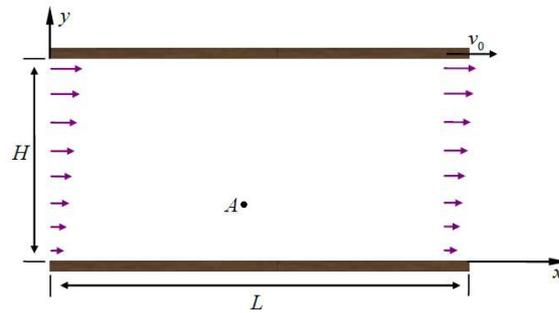
Figure 7: Flow between two parallel plates.

Table 3 lists the numerical errors of velocity u_x at the point $A(L/2,H/3)$ by using present SBM, compared with those by using the MFS, BEM and modified MFS (MMF-S) [22,23]. For the MFS solutions, the fictitious boundary has the same geometrical shape of the corresponding boundary. The distance d of the fictitious boundary away from the physical boundary is also quantified by the unit l which is the interval spacing of nodes on the physical boundary. The BEM solutions are obtained using the linear elements. The SBM, MFS and MMFS have the same boundary nodes distribution.

Table 3: Relative errors (RE) of the velocity u_x at the point $A(L/2,H/3)$.

N	Exact: 0.3333333E+01			
	RE (MFS $d=2l$)	RE (BEM)	RE (MMFS)	RE (SBM)
24	0.6666852E-03	0.8743401E-02	0.3847285E-01	-0.1406898E-01
40	-0.5184799E-03	0.3751984E-02	0.5442395E-02	-0.6769428E-02
80	-0.4128827E-01	0.9657298E-03	0.1307427E-02	-0.2717290E-02
160	-0.1767471E-02	0.4761407E-03	0.8622874E-03	-0.1140705E-02
320	-0.5977169E-04	0.1007384E-03	0.5429431E-03	-0.4900665E-03
640	-0.4058019E-03	0.7043791E-04	0.3042955E-03	-0.2109516E-03
1280	0.7539638E-03	0.5173159E-04	0.9575261E-04	-0.8984582E-04

From this table, we can observe that the present SBM, BEM and MMFS perform very well. The BEM solutions are slightly more accurate than those calculated by the SBM. But, the SBM is easier to program, and mathematically simple than the BEM. Compared with the MMFS results, the SBM solutions are less accurate with using small number of boundary nodes, and conversely are slightly more accurate when using more boundary nodes. It is also observed that the MFS solutions with the fictitious boundary $d=2l$ dont show convergence. This demonstrates that the location of the source nodes is vital to the accuracy and stability of the MFS solution.

Fig. 8 shows the comparison of tractions on boundary nodes with using the exact solution formulation and SBM formulation. Although only 48 boundary nodes are distributed on the boundary, we can see from the figure that satisfying results have been obtained.
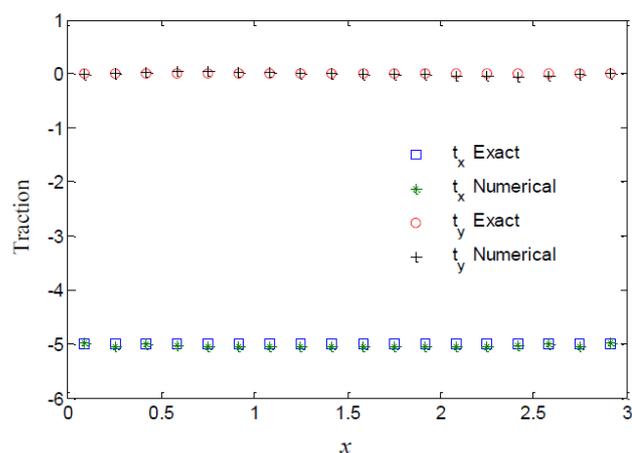
Figure 8: Traction profile at boundary nodes on the line $y=0$.

3.4   Flow through a channel with multiple cylinders

In this case, we consider a model with a 4×5 array of elliptic cylinders placed in the middle section of a channel $\{(x,y)|-1.5<x<1.5,\ -0.5<y<0.5\}$. The semi-major axis of every elliptic cylinder is $1/20$, and the semi-minor axis is $1/30$. At the inlet ($x=-1.5$) and outlet ($x=1.5$) of the channel, the flow has the following velocity profile

$$u_1=4v_0\left(\frac{1}{2}-y\right)\left(\frac{1}{2}+y\right),\quad u_2=0,\tag{3.3}$$

where $v_0=1$ is the maximum value of the velocity. No slip boundary conditions are assumed on the other boundaries. Note that the boundary nodes are uniformly distributed on the channel and every cylinder boundaries respectively.

  By using the SBM and BEM, Fig. 9 plots velocity solutions for a 4×5 array of elliptic cylinders with uniform and irregular distributions. The BEM solutions are obtained using the 600 linear elements (1200 boundary nodes), where channel and every cylinder boundaries are respectively discretised with 200 and 20 elements. For the SBM solutions, we also use 1200 boundary nodes, 400 of which on the channel boundary and 40 on every cylinder boundary. It can be observed that numerical results for the SBM agree pretty well with those for the BEM.

4   Conclusions

This paper proposes the SBM formulations for 2D Stokes flow problems. The origin intensity factors for the velocity, traction and pressure equations are derived, which is the key work of the SBM. The numerical solutions obtained with the present method agree well with the exact solutions. Furthermore, we can observe the numerical results exhibit
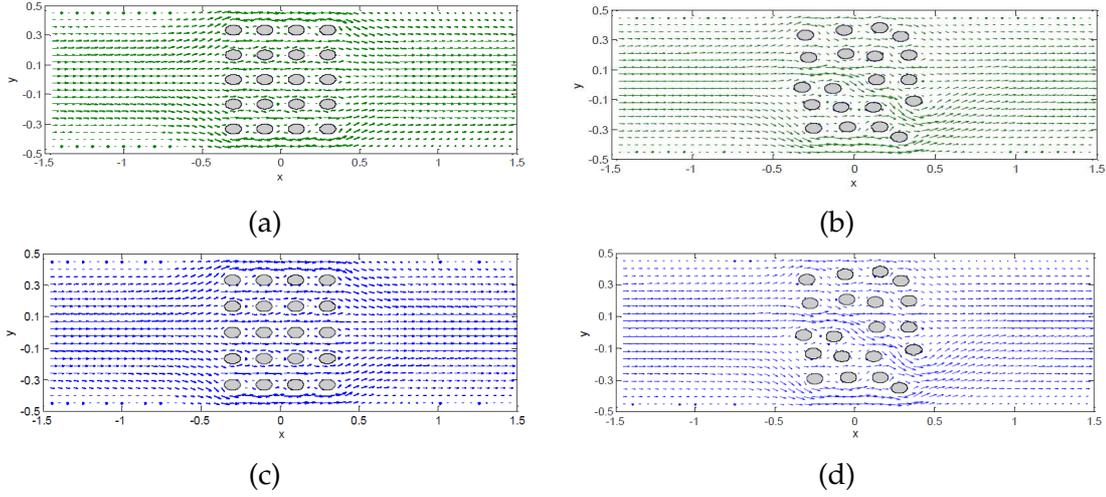
Figure 9: The velocity field of a channel with 20 elliptic cylinders by using the BEM: (a) for uniform distribution (b) for irregular distribution; by using the SBM: (c) for uniform distribution (d) for irregular distribution.

a stable convergence trend in all tested examples. All these demonstrate that the present SBM is efficient and accurate for the numerical analysis of 2D Stokes flow problems.

Appendix: Derivation of Eqs. (2.10) and (2.24)

According to the direct boundary integral equation [24], we have

$$\tau u_j(\mathbf{x}^m) = \int_\Gamma \left[U_{ij}(\mathbf{s}, \mathbf{x}^m) t_i(\mathbf{s}) - T_{ij}(\mathbf{s}, \mathbf{x}^m) u_i(\mathbf{s}) \right] d\Gamma(\mathbf{s}), \tag{A.1}$$

where $i, j = 1, 2$, u_i and t_i denote the velocity and traction, respectively, and

$$\tau = \begin{cases} 1, & \mathbf{x} \in \Omega, \\ 0, & \mathbf{x} \in \Omega^e. \end{cases} \tag{A.2}$$

By employing the simple test method ($t_1(\mathbf{s}) = t_2(\mathbf{s}) = 0$ when $u_1(\mathbf{s}) = 1$ and $u_2(\mathbf{s}) = 0$), we can write Eq. (A.1) as follows

$$\int_\Gamma T_{11}(\mathbf{s}, \mathbf{x}^m) d\Gamma(\mathbf{s}) = -\tau, \quad \int_\Gamma T_{12}(\mathbf{s}, \mathbf{x}^m) d\Gamma(\mathbf{s}) = 0. \tag{A.3}$$

In a similar way, by employing the simple test method ($t_1(\mathbf{s}) = t_2(\mathbf{s}) = 0$ when $u_1(\mathbf{s}) = 0$ and $u_2(\mathbf{s}) = 1$), we can write Eq. (A.1) as follows

$$\int_\Gamma T_{21}(\mathbf{s}, \mathbf{x}^m) d\Gamma(\mathbf{s}) = 0, \quad \int_\Gamma T_{22}(\mathbf{s}, \mathbf{x}^m) d\Gamma(\mathbf{s}) = -\tau. \tag{A.4}$$

We can rewrite Eqs. (A.3) and (A.4) as

$$\int_\Gamma T_{ij}(\mathbf{s},\mathbf{x}^m)d\Gamma(\mathbf{s}) = -\delta_{ij}\tau, \tag{A.5}$$

where δ_{ij} is the Kronecker delta, i.e.,

$$\int_\Gamma T_{ij}(\mathbf{s},\mathbf{x}^m)d\Gamma(\mathbf{s}) = -\delta_{ij}, \qquad \mathbf{x} \in \Omega, \tag{A.6a}$$

$$\int_\Gamma T_{ij}(\mathbf{s},\mathbf{x}^m)d\Gamma(\mathbf{s}) = 0, \qquad \mathbf{x} \in \Omega^e. \tag{A.6b}$$

When the field point $\mathbf{x}^m$ approaches the boundary, we can discretize Eq. (A.6b) as follows

$$\int_\Gamma T_{ij}^{\mathrm{E}}(\mathbf{s},\mathbf{x}^m)d\Gamma(\mathbf{s}) = \sum_{n=1}^N \int_{\Gamma_n} T_{ij}^{\mathrm{E}}(\mathbf{s},\mathbf{x}^m)d\Gamma(\mathbf{s}) \approx \sum_{n=1}^N L_n T_{ij}^{\mathrm{E}}(\mathbf{s},\mathbf{x}^m) = 0, \quad \mathbf{x}^m \in \Gamma, \tag{A.7}$$

where L_n is half distance between the source nodes $\mathbf{s}^{n-1}$ and $\mathbf{s}^{n+1}$. Thus, we have

$$\sum_{n=1}^N L_n T_{ij}^{\mathrm{E}}(\mathbf{s},\mathbf{x}^m) = 0, \quad \mathbf{x}^m \in \Gamma, \tag{A.8}$$

which is the Eq. (2.10) in Section 2.2.

Lemma A.1 (see [25]). *Assume Γ be a piecewise smooth curve, we have*

$$\int_\Gamma \frac{\partial \ln r(\mathbf{x}^m,\mathbf{s})}{\partial \mathbf{n}(\mathbf{s})}d\Gamma(\mathbf{s}) = \begin{cases} 2\pi, & \mathbf{x}^m \in \Omega, \\ 0, & \mathbf{x}^m \in \Omega^e, \end{cases} \tag{A.9a}$$

$$\int_\Gamma \frac{\partial \ln r(\mathbf{x}^m,\mathbf{s})}{\partial \mathbf{t}(\mathbf{s})}d\Gamma(\mathbf{s}) = 0, \quad \mathbf{x}^m \in \Omega \cup \Omega^e, \tag{A.9b}$$

where Γ denotes the boundary of the domain Ω as well as the outside domain Ω^e, $\mathbf{s}$ the source points, $\mathbf{x}^m$ the field point.

Based on the Lemma A.1, we can easily obtain the following equations

$$\int_\Gamma \frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s})}{\partial \mathbf{n}(\mathbf{s})}d\Gamma(\mathbf{s}) = 0, \qquad \int_\Gamma \frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s})}{\partial \mathbf{t}(\mathbf{s})}d\Gamma(\mathbf{s}) = 0, \qquad \mathbf{x}^m \in \Omega^e, \tag{A.10a}$$

$$\int_\Gamma \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s})}{\partial \mathbf{n}(\mathbf{s})}d\Gamma(\mathbf{s}) = 1, \qquad \int_\Gamma \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s})}{\partial \mathbf{t}(\mathbf{s})}d\Gamma(\mathbf{s}) = 0, \qquad \mathbf{x}^m \in \Omega. \tag{A.10b}$$

In a similar way, based on the (A.10a) and (A.10b), we have

$$\sum_{n=1}^N L_n \frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)} = 0, \qquad \sum_{n=1}^N L_n \frac{\partial \psi^{\mathrm{E}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)} = 0, \qquad \mathbf{x}^m \in \Gamma, \tag{A.11a}$$

$$\sum_{n=1}^N L_n \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{n}(\mathbf{s}^n)} = 1, \qquad \sum_{n=1}^N L_n \frac{\partial \psi^{\mathrm{I}}(\mathbf{x}^m,\mathbf{s}^n)}{\partial \mathbf{t}(\mathbf{s}^n)} = 0, \qquad \mathbf{x}^m \in \Gamma, \tag{A.11b}$$

where Eq. (A.11a) is the Eq. (2.24) in the previous Section 2.3. Besides, the Eq. (A.11b) can be used for the derivation of Eq. (2.27).

Acknowledgements

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